

Design & Analysis Of Processor Cooling Fins

Madhu B P¹, Kamal prashanth T², Kevin Jeffrey A³, Suraj N⁴, Vashantha kumar J⁵

^{1,2,3,4,5}Dept of Mechanical engineering

^{1,2,3,4,5}REVA University.

Abstract- In this study, thermal analysis of electronic processor fins is proposed and an effort is made to understand temperature distribution in processor by employing rectangular fins which aid in rapid heat removal to the surroundings for ensuring the optimal working of the processor. Removal of heat generated in the processor gets augmented by the application of fins to it.. Modeling is done in CATIA V5 and Analysis is carried out using Multi-physics software, COMSOL. Heat flows out from the processor to the surrounding through the casing and then to fins attached to it. Convective boundary condition is applied to one face of a fin.. The results report the temperature distribution and heat transfer rate contour for variation in fin length. Results show that COMSOL can be used effectively and efficiently to solve the challenge of heat transfer problem.

Keywords- CATIA V5, COMSOL ,Metal fin, Thermal analysis.

I. INTRODUCTION

We need to go far back in time to remember a CPU that was able to operate completely without a heat sink. The first Intel processors were already producing considerable amount of heat, but the low specifications allowed operation without any heat removal mechanism. A little later, as the processing speed increased, these processors required at least a passive heat sink for trouble free operation. However, for the last few years, as the processors got more and more powerful, it has become mandatory that a CPU requires a multi-fin heat sink as well as a fan that ensures reasonable air flow through the cooling fins as the overheated processors exhibit a shorter maximum life span and often results in problems like system freezes or crashes. A heat sink is a device used in computers to remove the large amount of heat generated by components, including ICs such as CPUs, chipsets and graphic cards, during their operation. A heat sink is used to increase the surface area which dissipates the heat faster n keeps the ICs under safe operating temperature. Fans are also used to speed up this process. It usually consists of a base with one or more flat surfaces and an array of fin like protrusions to increase the heat sink's surface area contacting the air, and thus increasing the heat dissipating rate. A combination of a heat sink and a fan is widely used which maintains a larger temperature gradient by replacing warmed air more quickly. Heat sinks are

made from good thermal conductors such as copper or aluminum alloy. Copper is significantly heavier and more expensive than aluminum but it is also roughly twice as efficient. The most common of a heat sink is a metal device (Cu or Al) with many fins. In this paper, section 2 describes the heat sink types and its thermal resistance. Section 3 explains the least square model for estimating the parameters and predicting the best fit of the proposed heat sink in different loads of a microprocessor. In electronic systems, a heat sink is a passive heat exchanger that cools a device by dissipating heat into the surrounding medium. In computers, heat sinks are used to cool central processing units or graphics processors. Heat sinks are used with high-power semiconductor devices such as power transistors and optoelectronics such as lasers and light emitting diodes (LEDs), where the heat dissipation ability of the basic device is insufficient to moderate its temperature. A heat sink is designed to maximize its surface area in contact with the cooling medium surrounding it, such as the air. Air velocity, choice of material, protrusion design and surface treatment are factors that affect the performance of a heat sink. Heat sink attachment methods and thermal interface materials also affect the die temperature of the integrated circuit. Thermal adhesive or thermal grease improve the heat sink's performance by filling air gaps between the heat sink and the heat spreader on the device.

A. Heat transfer principal

A heat sink transfers thermal energy from a higher temperature device to a lower temperature fluid medium. The fluid medium is frequently air, but can also be water, refrigerants or oil. If the fluid medium is water, the heat sink is frequently called a cold plate. In thermodynamics a heat sink is a heat reservoir that can absorb an arbitrary amount of heat without significantly changing temperature. Practical heat sinks for electronic devices must have a temperature higher than the surroundings to transfer heat by convection, radiation, and conduction. The power supplies of electronics are not 100% efficient, so extra heat is produced that may be detrimental to the function of the device. As such, a heat sink is included in the design to disperse heat to improve efficient energy use. Fig. 1 shows how the heat sink looks. With the same image principle of heat transfer can be understood easily.

To understand the principle of a heat sink, consider Fourier's law of heat conduction. Fourier's law of heat conduction, simplified to a one-dimensional form in the x -direction, shows that when there is a temperature gradient in a body, heat will be transferred from the higher temperature region to the lower temperature region.

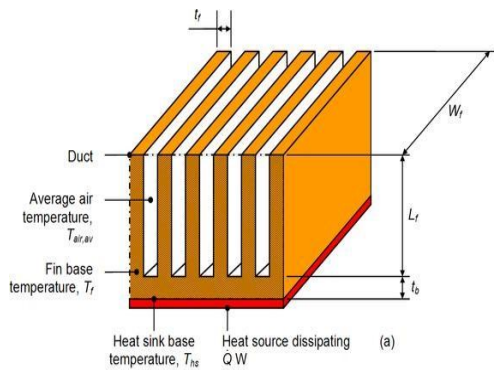


Fig. 1 heat transfer principal

The rate at which heat is transferred by conduction, q_k , is proportional to the product of the temperature gradient and the cross-sectional area through which heat is transferred.

$$q_k = -kA \frac{dT}{dx}$$

Consider a heat sink in a duct, where air flows through the duct, as shown in Figure. It is assumed that the heat sink base is higher in temperature than the air. Applying the conservation of energy, for steady-state conditions, and Newton's law of cooling to the temperature nodes gives the following set of equations.

$$\begin{aligned} \dot{Q} &= \dot{m}c_{p,air}(T_{air,out} - T_{air,in}) \quad (1) \\ \dot{Q} &= \frac{T_{hs} - T_{air,av}}{R_{hs}} \quad (2) \end{aligned}$$

B. Design factors to be considered

While designing fins factors to be considered are thermal resistance, fin efficiency, spreading resistance, optimization, fin arrangement and surface color.

II. LITERATURE

Knight et al. (1992) extended previous analysis of micro channel heat sinks for turbulent as well as laminar flow. They demonstrated improvement of previous studies by relaxing constraints on fin thickness/ pitch ratio and allowing turbulent flow. Copeland (1995) modified previous analyses

for developing flow and calculated optimum fin thickness and pitch for silicon heat sinks cooled by fluorocarbon liquids. Lee (1995) analysed flow through parallel fin heat sinks in fully ducted and partially ducted flows. Unlike a fully ducted configuration, in partially ducted configuration at a fixed approach velocity, an optimum size of fin existed; thermal performance improves monotonically as fin pitch is decreased. Aranyosi et al. (1997) showed isocurves of pressure drop and fan power at fixed thermal resistance in addition to isocurves of thermal resistance at fixed pressure drop and fan power. As pressure drop or fan/blower power increased, optimum fan thickness and pitch decreased, resulting in reduced thermal resistance. In addition to analysis, experimental and numerical studies were performed. Tasaka et al. (1997) performed experimental studies of compact heat sinks with fin thickness and pitch as small as 0.34mm and 0.70mm. Results correlated well with results from compact heat exchanger data. This compactness factor, defined as thermal conductance per unit volume, was three to seven times that of standard heat sinks. M Bisht and K S Mehra (2014) conducted number of experiments and published a paper on optimisation of working of processor by fins using FEM in International Journal for Research in Applied Science and Engineering Technology.

III. PROBLEM SPECIFICATION

The heat sink of processor is made up of Aluminium of thermal conductivity of 167W/m-K. Processor generates heat of 54 W. There is convection along all the boundaries except the bottom, which is insulated. The film (convection) coefficient is $h=20 \text{ W/m}^2\text{-K}$ and the ambient temperature is 25°C. An attempt is done to understand the heat transfer rate in processor. The properties of Aluminium material are:

- Thermal conductivity : 167 W/m-K
- Melting point : 660 °C
- Specific heat : 0.9 J/g ·°C
- Thermal diffusivity : 9.5e-7 m²/s
- Thermal expansion : 23/°C

IV. PROCESS METHODOLOGY

The process of solving this in FEM by using COMSOL is as below

A. Solid modeling

Solid modeling is done in CATIA V5 and converted into .stp file format so that we can import this model easily in COMSOL 15. Solid modeling is as shown in the below figure that is Fig. 2.

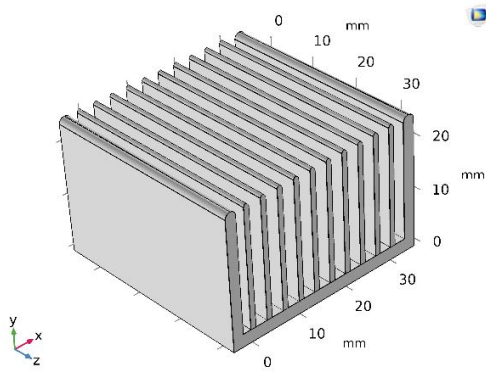


Fig.2 Solid model

B. Static thermal analysis of metal fins.

The temperature distribution of metal fin is obtained from static thermal analysis from the following steps:

- 1) Designer module: The model of metal fin is imported into COMSOL designer module as step file (.stp), where the cleaning and the stitching of the model surfaces is done.
- 2) Meshing module : The finite element mesh is used to subdivide the CAD model into smaller domains called elements, over which a set of equations are solved. Fig 4.14 shows the meshed model of Fin.

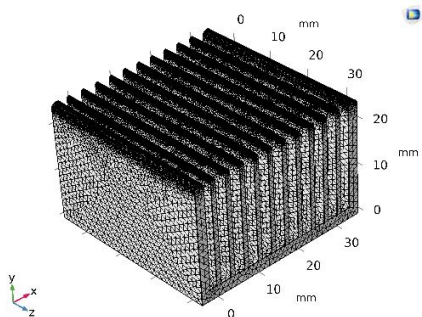


Fig.3 Meshed model

- 3) Applying the boundary conditions: Boundary condition is the one that is required to be satisfied at all or part of the boundary of a region in which a set of differential conditions is to be solved.
 - Ambient temperature : 25°C
 - Boundary heat source : 1W
 - Heat transfer coefficient : 50 W/m²-k

C. Solution

It includes temperature distribution and heat transfer rate contours ; here we specify the constraints and finally solve the resulting set of equations. In this problem one side of

block have convection type of boundary condition except the bottom side which is insulated and analysis type is Steady state thermal analysis.

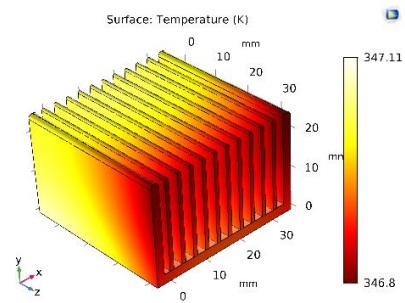


Fig.4 Temperature contour

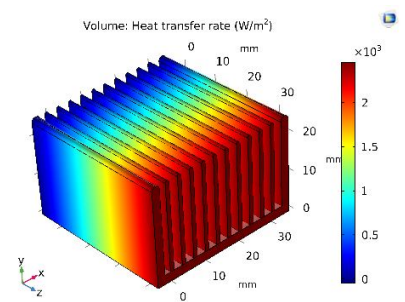


Fig 5. Heat transfer con

V. RESULTS

Temperature and heat transfer rate contours are plotted for the whole fin. Results are summarized in the form of graph for Maximum Temperature and Heat transfer rate for different cases of fins.. Results are summarized in the form of graph for Temperature and Heat transfer rate for whole fin.

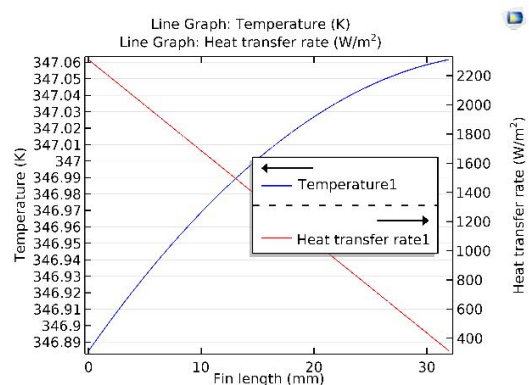


Fig. 6 Variation of temperature and heat transfer rate with respect to fin length.

VI. CONCLUSION

Thermal analysis of processor is done for ensuring its optimal working. Rectangular fins of aluminum are attached to the steel casing enclosing the processor. The results have been obtained by increasing the number of fins. The temperature distribution and heat flux contours with different selection in number of processor fins are obtained. It is observed that the heat transfer rate is decreasing and temperature is increasing. But after 30mm fin length there is no considerable increase in temperature.

VII. FUTURE SCOPE

- Thermal analysis for processor cooling system with different shapes of fins and optimization of the same.
- Cost reduction analysis with variable parameters.
- CFD analysis of the fins, to understand air flow around the fins.

REFERENCES

- [1] R.L.Linton and D. Agonafer, "Thermal model of a PC", ASME Journal of Electronic Packaging, vol. 116, pp.134-137, 1994.
- [2] R. J. Yang, and L. M. Fu, "Thermal and flow analysis of a heated electronic component," International journal of heat and mass transfer, vol. 44, pp. 2261-2275, 2001.
- [3] C. W. Yu, and R. L. Webb, "Thermal design of a desktop computer system using CFD analysis", Seventeenth IEEE SEMI- THERM SYMPOSIUM, pp. 18-26, 2001.
- [3] J. Y. Chang, "Identification of minimum air flow design for a Desktop computer using CFD modeling", " 7th intersociety conference on thermal and thermo mechanical Phenomena in Electronic systems", vol. 1, pp.330-338, 2000.
- [4] D.Lober, "Optimizing the integration of an electronics system into an existing enclosure using CFD modeling techniques", International journal of microcircuits and electronic packaging, vol. 22, pp.146-151, 1999.
- [5] S. Subramanyam, and K.E.Crowe, "Rapid design of heat sinks for electronic cooling computational and experimental tools", IEEE Symposium, pp. 243-251, 2000.
- [6] J. H. Ryu, D. H. Choi, and S. J. Kim, "Numerical Optimization of the thermal performance of a micro channel heat sink", Int. J. Heat and Mass Transfer, Vol.45, pp.2823-2827, 2002.
- [7] R. W. Knight, D. J. Hall, J. S. Goodling, and R. C. Jaeger,
- [8] "Heat Sink Optimization with Application to Micro channels", IEEE Transactions on Components, Hybrids, and Manufacturing Technology, vol. 15, no. 5, pp. 832-842, 1992.
- [9] T. C. Hung, S. K. Wangi, and F. Peter, "Simulation of Passively Enhanced Conjugate Heat Transfer Across an Array of Volumetric Heat Sources" Communications in Numerical Methods in Engineering, John Wiley & Sons, Ltd. vol.13, pp. 855-866, 1997.
- [10] Egan, Eric, Amon, and H. Cristina, "Thermal Management Strategies for Embedded Electronic Components of Wearable Computers", Journal of Electronic Packaging, vol.122, pp. 98-106, June 2000.
- [11] Rodgers, J. Peter, Evely, C. Valerie, Davies, and R. D. Mark,
- [12] " An Experimental Assessment of Numerical Predictive Accuracy for Electronic Component Heat Transfer in Forced Convection-Part 2, Results and Discussions", Journal of Electronic Packaging, ASME, vol.125, pp. 76-83, March 2003. [12] Lorenzini, Giulio., Biserni, and Cesare, "A Vepotron Effect Application for Electronic Equipment Cooling", Journal of Electronic Packaging, ASME, vol.125, pp. 475-479, Dec 2003.
- [13] Davies, R. D. Mark, Cole, Reena, Lohan, and John, "Factors Affecting the Operational Thermal Resistance of Electronic Components", Journal of Electronic Packaging, ASME, vol. 122, pp. 185-191, Sep 2000.
- [14] F. P. Incropera, and D. P. DeWitt, Fundamentals of Heat and Mass Transfer, 5th Ed, Wiley.