Enhancement of Electromagnetic Braking System

Sanket Solim¹, Vinayak Parab², Tushar Shinde³, Akshay Surve⁴, Prof. Sumit malusare⁵

^{1, 2, 3, 4, 5} Dept of Mechanical ^{1, 2, 3, 4, 5} FAMT Ratnagiri

Abstract- We know that today's braking system works on principle of dissipation of kinetic energy into heat energy that is through friction.[1] As this method has wearing effects hence must be replaced with a method which has low or no wearing effect with enhanced effectiveness.[2] In this project design of electromagnetic braking system along with optimization for various operational parameters has been done. These parameters have been previously iterated in cited projects and papers. Simulation models are to be cross checked with experimental setup. An attempt is made through this project to calculate the braking time.

Keywords- Current, braking torque, opposing torque, no. of turns, air gap

I. INTRODUCTION

Modern braking techniques focused on safety, comfort and environmental protection. With the increase in driving speed and load conventional braking systems does not give satisfactory performance.[4] Electromagnetic brakes can be applied separately, completely without the use of friction brakes. Due their specific method of installation electromagnetic brakes can avoid problems that friction brakes face. Working principle of electric retarder (solenoid ring) is based on creation of eddy current within a coil which is mounted parallel to flywheel. The force developed by eddy current opposes the flywheel. [6]

Therefore, practically it is found that electromagnetic brakes depend on electrical parameters like flux density, axial distance between solenoid coil, current etc. This electromagnetic braking system eliminates drawback of wear and tear which exist in conventional braking system. This system is applicable for only high speed application.

II. METHODOLOGY

Initially designing of setup is done with help of solid works. Then calculations regarding various components involved in it like shaft, flywheel etc are done. So the method involves taking actual readings over the setup and then comparing it with theoretical values which are calculated with the help of standard formulas.

CAD Model:-





Flywheel:-



Fig no 2.2 Flywheel

Total weight of flywheel (M) = 8.94kg Weight of flywheel with 6 holes= 7.95kg Radius of flywheel (R)= 152mm Radius of holes (r) = 18.5mm Distance of centre of hole from distance of flywheel (h) = 95mm Mass moment of inertia of flywheel without hole (I) = MR²/2 =0.10327 kg.m² Mass moment of inertia of holes (Ih) = Io+m.h² = (mr²/2)+m*h² =1.50816*10⁻³ kg.m²

 $I_{flywheel} = I - I_h$ $= 0.09422 kg.m^2$

Solenoid ring:-



Fig no.2.3:- Solenoid ring

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Details of solenoid ring:-

- 1) Material: Carbon steel
- 2) Permeability:- $1.26*10^{-4}$ H/m
- 3) Relative permeability: 100
- 4) Copper wire diameter: 0.5 mm
- 5) No. Of turns: 1500

Actual setup:-



Fig no.2.4:- Experimental setup

Experiments are performed on above setup which comprises D.C. motor (0.5KW) to which flywheel is connected with the help of shaft. A braking arrangement is shown in fig 2.3 which is termed as solenoid ring. To supply controlled voltage dimmer stat is connected to dc motor as well as to solenoid ring but where ammeter is connected to it in series. Experiments are performed on setup within the range of 400 to 1100 RPM with 5mm and 3 mm air gap.

III. PRACTICAL OBSERVATIONS

Terms involved:-

Tactw = Actual time without braking

Tactb = Actual time with braking

Tthw = Theoretical time without braking

Tthb = Theoretical time with braking

 ζ coil = Opposing torque by solenoid ring = 0.1576 N.m

 ζ Motor = Torque of motor

 αw and αw = Deceleration of flywheel with and without braking

Table no. 3.1 For 3mm air gap and 5 A current

Tactw(sTactb(s_Motor aw(rad/Tthw(seab(rad/se Tthb										
RPM)	(N.m)	sec ²)	c)	c ²)	(sec)			
1100	34.59	31.34	4.34	46.062	23.88	47.73	23.04			
1000	30.37	30.11	4.77	50.62	19.75	52.29	19.12			
900	27.82	26.7	5.3	56.25	15.99	57.9	15.53			
800	25.53	25.13	5.96	63.25	12.64	64.92	12.32			
700	23.1	22.92	6.82	72.38	9.67	74.05	9.45			
600	20.8	19.47	7.95	84.37	7.11	86.04	6.97			
500	17.1	15.93	9.54	101.25	4.93	102.92	4.8			
400	13.73	13.43	11.93	126.6	3.15	128.29	3.11			



Graph 3.1:- For actual reading with and without braking

The above graph shows the variation of time with respect to RPM , with and without braking for practical readings for 3mm air gap.



Graph 3.2:- For theoretical reading with and without braking

The above graph shows the variation of time with respect to RPM, with and without braking for theoretical readings for 3 mm air gap.

Table No. 3.2 :- For 5mm air gap and 5A current

		<u> </u>						
			Motor	aw(rad/s'		ab(rad/s		
RPM	Tactw(s)	Tactb(s)	(N.m))	Tthw(s)	²)	Tthb (sec)	
1100	34.59	32.75	4.34	46.06	23.88	47.73	23.04	
1000	30.37	30.23	4.77	50.62	19.75	52.29	19.12	
900	27.82	26.83	5.3	56.25	15.99	57.94	15.53	
800	25.53	25.46	5.96	63.25	12.64	64.92	12.32	
700	23.1	23.03	6.82	72.38	9.670	74.05	9.45	
600	20.8	19.78	7.95	84.37	7.110	86.04	6.97	
500	17.1	16.08	9.54	101.25	4.93	102.9	4.85	
400	13.73	13.4	11.9	126.61	3.15	128.1	3.11	



Graph 3.3 For Actual reading with and without braking

The above graph shows the variation of time with respect to RPM, with and without braking for practical readings 5 mm air gap.



Graph 3.4 :-For theoretical reading with and without braking

The above graph shows the variation of time with respect to RPM, with and without braking for theoretical readings for 5mm air gap

IV. CONCLUSION

The purpose of study was to perform a comparative study of practical and theoretical braking time and establish practical air gap limit beyond which the electromagnetic brakes loses their effectiveness.

From theoretical calculation and experimental braking time maximum air gap limit of 3mm is obtained beyond which electromagnetic brakes are found to be ineffective. It is also found that the system is efficient for higher range of RPM (800 -1100).

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V. FUTURE SCOPE

If solenoid rings are incorporated on both sides of flywheel then the braking effect would increase. It would be possible to go for different kind of materials to check braking effect. Also magnets can be positioned around the flywheel to get better braking torque distribution.

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